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Aerodynamic Analysis Models for Vertical-Axis Wind Turbines

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This work details the progress made in the development of aerodynamic models for studying Vertical-Axis Wind Turbines (VAWT's) with particular emphasis on the prediction of aerodynamic loads and rotor performance as well as dynamic stall simulations. The paper describes current effort and some important findings using streamtube models, 3-D viscous model, stochastic wind model and numerical simulations of the flow around the turbine blades. Comparison of the analytical results with available experimental data have shown good agreement.

Key Words: Wind turbines; Aerodynamics; Atmospheric turbulence; Dynamic stall; Navier-Stokes equations.

Through advanced technology, wind turbine has become an important commercial option for largescale power production. As a result, there are now more than 2000 megawatts of wind power in the world, most of them in the US and Denmark. Modern wind turbines can basically be classified as either of vertical-axis design, such as the Darrieus model, or as horizontal-axis variety like the traditional farm windpump. In both cases, the turbine's rotating blades extract kinetic energy from the wind to generate electricity or pump water. The vertical-axis wind turbine, such as Darrieus model with curved blades offers an advantageous alternative to the horizontal axis wind turbine due to its mechanical and structurally simplicity of harnessing the wind energy. This simplicity, however, does not extend to the rotor's aerodynamic since the blade elements encounter their own wakes and those generated by other elements and operate in a dynamic stall regime (Brochier et al., 1986). Added to this is the increasing awareness that atmospheric turbulence and fluctuating loads significantly affect the turbine output (Turyan et al., 1987). The J.A. Bombardier Aeronautical Chair Group at École Polytechnique de Montréal has conducted many research on the development of computer codes for studying Darrieus rotor aerodynamics (Paraschivoiu, 1988).

The objective of the computer programs is to determine aerodynamic forces and power output of the vertical-axis wind turbine of any geometry at a chosen rotational speed and ambient as well as turbulent wind. Three computer code variants based on the doublemultiple streamtube model, stochastic wind and viscous flow field have been developed. The 3DVF viscous flow model based on Navier-Stokes equations analyses the Darrieus rotor in a steady incompressible laminar flow field by solving the Navier-Stokes equations in a cylindrical coordinates with the finite volume method where the conservation of mass and momentum are solved by using the primitive variables p, u, v, and w (Allet et al., 1992). Since the ambient wind has been considered to be constant the predicted loads on the blades are identical for each rotor revolution. In order to take the fluctuating nature of the wind into account a 3-D stochastic model has been developed and incorporated first into the double-multiple streamtube model to analyse the effect of atmospheric turbulence on aerodynamic loads (Brahimi et al. 1992). Actually more work are underway to incorporate the stochastic wind into the 3DVF viscous model. For the dynamic stall simulation the computer codes developed use empirical model. Although the empirical dynamic stall models used predict well the aerodynamic loads and rotor performance, they are limited to the type of airfoil and motion used in the experiment from which they were derived. Thus, a new code based on the Navier-Stokes equations and uses the finite element method for simulating the dynamic stall around Darrieus wind turbine has been developed (Tchon et al., 1993). Since 3-D simulation would be very expensive a 2-D simulation has been adopted. The model uses a non-inertial stream function-vorticity formulation (Ψ,ω) of the 2-D incompressible unsteady Navier-Stokes equations. The computer code was first validated for the flow around a rotating cylinder then, it was applied to simulate the flow around a NACA 0015 airfoil in Darrieus motion. The present paper presents some development of the streamtube models, the 3-D viscous model, the stochastic wind as well as the numerical simulation of dynamic stall.

MOMENTUM MODELS

Several aerodynamic prediction models currently exist for studying Darrieus wind turbines and a complete state of the art review including the appropriate references is given by Strickland (1986) and Paraschivoiu (1988). Generally, the main objective of all aerodynamic models is to evaluate the induced velocity field of the turbine since knowledge of this velocity field allows all the forces on the blade and the power generated by the turbine to be determined.

The first approach to analyze the flow field around vertical-axis wind turbine was developed by Templin (1974) who considered the rotor as an actuator disk enclosed in a simple streamtube where the induced velocity through the swept volume of the turbine is assumed to be constant. An extension of this method to the multiple-streamtube model was then developed by Strickland (1975) who considered the swept volume of the turbine as a series of adjacent streamtubes. Other aerodynamic methods for modeling the wind turbine are based on the vortex theory (Strickland, et al., 1980). Basically two types of the vortex model have been used: the fixed-wake and the free-wake models. Although these models have the advantage to predict the aerodynamic loads and performance more exactly than the momentum models, they require a considerable amount of computer time. Paraschivoiu (1981) developed an analytical model (DMS) that considers a multiple-streamtube system divided into two parts where the upwind and downwind components of the induced velocities at each level of the rotor are calculated by using the principle of two actuator disks in tandem. Three categories of computer codes have been developed (Fig. 1): *CARDAA* which uses two constant interference factors in the induced velocities calculated by a double iteration, *CARDAAV* code which considers the variation of the interference factors as a function of the azimuth and *CARDAAX* code which takes the streamtube expansion into account. These codes have been used at IREQ, Sandia National Laboratories, DAF Indal Co., IMST Marseilles and elsewhere.

For the upstream half-cycle of the rotor, the relative velocity and the local angle of attack as a function of tip speed ration "X" are given by:

$$W^{2} = V^{2} \left[(X - \sin \theta)^{2} + \cos^{2} \theta \cos^{2} \delta \right]$$
 (1)

$$\alpha = \arcsin \left[\frac{\cos \theta \cos \delta}{\sqrt{(X - \sin \theta)^2 + \cos^2 \theta \cos^2 \delta}} \right]$$
(2)

The normal and tangential forces coefficients are evaluated for each streamtube as a function of the blade position using the blade airfoil sectional force coefficients:

$$C_N = C_L \cos\alpha + C_D \sin\alpha \tag{3}$$

$$C_T = C_L sin\alpha - C_D cos\alpha \tag{4}$$

where the blade airfoil section lift and drag coefficients, C_L and C_D , are obtained by interpolating the available test data using both the local Reynolds number ($Re_b = W c/\nu$) and the local angle of attack.

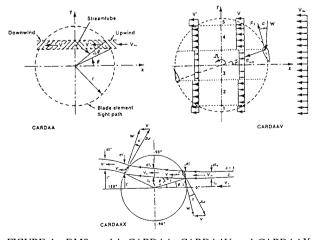


FIGURE 1 $\,$ DMS model; CARDAA, CARDAAV, and CARDAAX computer codes.

STOCHASTIC WIND MODEL

The earlier aerodynamic models for studying Vertical Axis Wind Turbine (VAWT) are based on a constant incident wind conditions and are thus capable of predicting only periodic variations in the loads (Veers, 1984), (Malcolm, 1987), (Marchand *et al.*, 1987), (Strickland, 1987) and (Homicz, 1988). As a result, the predicted loads on the blade are identical for each rotor revolution and we have no information about the effect of turbulence on the rotor. Indeed the atmospheric turbulence as seen by rotating wind turbines has become an important factor for studying stochastic aerodynamic loads and turbulent flow effects have been identified as one of the major source of rotor blade fatigue life.

Continuing the development of the *DMS* model, a new code (*CARDAAS*) has been developed to predict loads on VAWT by taking the fluctuating nature of the wind into account. The velocity field of the wind is assumed to be a linear superposition of a steady or mean component and a fluctuating component. The main objective of the wind model is to simulate the turbulent velocity fluctuations. It includes both the streamwise and lateral component of the turbulent velocity. The one dimensional variations of this turbulent wind are introduced by creating time series of the wind velocity at a fixed point upwind the rotor and assuming that the wind speed is constant in a plane perpendicular to the mean wind direction.

The turbulent wind speed downstream of the fixed point is obtained by calculating a time delay in the time series. The decrease in the streamwise velocity as the flow passes through the rotor is taken into account by assuming a linear variation in the streamwise direction. The method used for the 3-D wind model is to simulate wind speed time series at several points in the plane perpendicular to the mean wind direction (Fig. 2). For

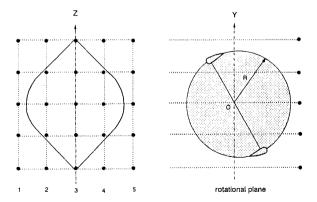


FIGURE 2 Schematic of 3-D wind simulation.

each point the time series is generated to represent the variation about the mean velocity in the longitudinal and vertical directions. The relative velocity and the local angle of attack are:

$$W^{2} = [\Omega r - (V + u_{f})\sin\theta - v_{f}\cos\theta]^{2} + [(V + u_{f})\cos\theta - v_{f}\sin\theta]^{2}\cos^{2}\delta$$
 (5)

$$\alpha = \arcsin\left[\left(\left(V + u_f\right)\cos\theta - v_f\sin\theta\right)\cos\delta\right]/W \quad (6)$$

where Ω represents the turbine rotational speed, r the local rotor radius, V the induced velocity for each streamtube as a function of the azimuthal angle θ , u_f and v_f the fluctuating velocities and δ the meridional angle.

The fluctuation velocities due to the turbulent wind are represented by a Fourier time-series (Brahimi, 1992):

$$V_f^+ = \sigma^+ \sum_{j=1}^{N_p/2} \left[A_j^+ \sin(2\pi \eta_j^+ \tau^+) + B_j^+ \cos(2\pi \eta_j^+ \tau^+) \right]$$
(7)

where V_f^+ represents the normalized fluctuation velocities, $V_f^+ = V_f / V_{\infty i}$ ($V_f = u_f, v_f$), and σ^+ represents the normalized turbulence intensity, $\sigma^+ = \sigma / V_{\infty i}$. The unknown Fourier coefficients A_j^+ and B_j^+ are given in terms of the non dimensional spectral power density, Φ_i the dimensionless frequency band $\Delta \eta^+$ and a random phase angle ϕ_i by:

$$A_i^+ = (2\Phi_i \Delta \eta^+)^{1/2} \sin(\phi_i)$$
 (8)

$$B_i^+ = (2\Phi_i \Delta \eta^+)^{1/2} \cos(\phi_i)$$
 (9)

The fluctuation velocities are performed by using a Fast Fourier Transform, then the aerodynamic loads are evaluated for each streamtube as function of the blade positions using the total flow velocities.

THREE-DIMENSIONAL VISCOUS MODEL

Since the *DMS* codes do not take the viscous effects into account a computer code named 3DVF (Allet, 1993) has been developed. This code analyses the Darrieus rotor (Fig. 3) in a steady incompressible laminar flow field by solving the Navier-Stokes equations in a cylindrical coordinates using the finite volume method. The conservation of mass and momentum are solved using the primitive variables p, u, v, and w. The effect of the spinning blades is simulated by distributing source terms in the ring of control volumes that lie in the path of the turbine blades.

of the turbine.

By using Fig. 3 the relative velocity \overrightarrow{V}_{rel} is calculated by the following equation:

$$\overrightarrow{V}_{rel} = V_n \overrightarrow{e}_n + \overrightarrow{V}_{\theta}' \overrightarrow{e}_{\theta} = (\overrightarrow{V}_{abs} \overrightarrow{e}_n) \overrightarrow{e}_n + (-\Omega r + \overrightarrow{V}_{abs} \overrightarrow{e}_{\theta}) \overrightarrow{e}_{\theta}$$
(10)

The only unknown parameters in the above equation (Eq. (10)) is the absolute velocity, \overrightarrow{V}_{abs} , which is computed by using the Navier-Stokes equations. The discretization method used to solve the governing equations is based on the finite volume method (FVM) proposed by Patankar (1980). For velocity-pressure coupled flows, a staggered grid system is known to give more realistic solutions and is adopted in the present study. The calculating method based on the control volume approach used here is the widely known "SIMPLER" algorithm. Details of the governing equations with the numerical procedure are given by Allet (1993). The motion of the Darrieus blades are time averaged and introduced through the source

$$S_r = Nc\rho V_{rel} \frac{\Delta\theta}{2\pi} \cos\delta (C_D u \cos\delta - C_L V_\theta' + C_D w \sin\delta) (11)$$

terms into the momentum equations. The source terms

are valid for all the computational cells that lie in the path

$$S_{\theta} = Nc\rho V_{rel} \frac{\Delta \theta}{2\pi} (C_D \nu + C_L V_n - C_D \Omega r) \qquad (12)$$

$$S_z = Nc\rho V_{rel} \frac{\Delta\theta}{2\pi} sin\delta(C_D w sin\delta - C_L V_\theta' + C_D u cos\delta)$$
(13)

All forces and power computations are integrated for the grid points that lie in the path of the blades.

DYNAMIC STALL SIMULATION

Dynamic stall is an unsteady flow phenomenon which refers to the stalling behavior of an airfoil when the angle

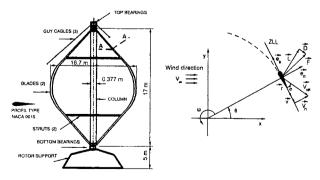


FIGURE 3 Angles, velocity vectors and forces for Sandia 17-m VAWT.

of attack is changing rapidly with time. It is characterized by dynamic delay of stall to angles significantly beyond the static stall angle and by massive recirculating regions moving downstream over the airfoil surface.

In the case of Darrieus wind turbine, when the operational speed approaches its maximum, all the blade sections exceed the static stall angle, the angle of attack changes rapidly and the whole blade operates under dynamic stall conditions. This increases the unsteady blade loads and structural fatigue (Ham, 1967, Philippe et al., 1973, Gormont, 1973, McCroskey et al., 1976 and McCrosky, 1981). Semi-empirical dynamic stall models have already been included in our computer codes based on DMS models as well as on the 3-D viscous model. These are namely the Gormont model (Gormont, 1973), MIT model and Indicial model (Paraschivoiu et al., 1988, Proulx et al., 1989). Although the dynamic stall models predict well the aerodynamic loads and performance on Darrieus wind turbine, they are limited to the type of airfoil and motion used in the experiment from which they were derived.

A new code for simulating the dynamic stall around Darrieus wind turbine has been developed, it is called "TKFLOW" (Tchon et al., 1993). This code is based on the Navier-Stokes equations and uses the finite element method. Since 3-D simulation would be very expensive a 2-D simulation has been adopted. The model uses a non-inertial stream function-vorticity formulation (Ψ , ω) of the 2-D incompressible unsteady Navier-Stokes equations. The computer code was first validated for the flow around a rotating cylinder (Tchon et al., 1990). Then, it was applied to simulate the flow around a NACA 0015 airfoil in Darrieus motion.

The vorticity transport equation and the stream function compatibility are given by:

$$\frac{\partial \omega}{\partial t} + \nabla \cdot [\omega \overrightarrow{u} - \nabla \nu_e \omega + 2S_{\omega}] \tag{14}$$

$$S_{\omega} = \left\{ \begin{array}{l} \frac{\partial \nu_{e}}{\partial \varkappa_{1}} \frac{\partial^{2} \Psi}{\partial \varkappa_{2}^{2}} - \frac{\partial \nu_{e}}{\partial \varkappa_{2}} \frac{\partial^{2} \Psi}{\partial \varkappa_{1} \partial \varkappa_{2}} \\ \frac{\partial \nu_{e}}{\partial \varkappa_{2}} \frac{\partial^{2} \Psi}{\partial \varkappa_{2}^{2}} - \frac{\partial \nu_{e}}{\partial \varkappa_{1}} \frac{\partial^{2} \Psi}{\partial \varkappa_{1} \partial \varkappa_{2} \partial \varkappa_{2}} \end{array} \right\}$$

The stream function compatibility equation is:

$$\nabla^2 \Psi + \omega = 0 \tag{15}$$

The effective viscosity is given by $v_e = v + v_t$, where v_t represents the eddy viscosity. The computational mesh used in the TKFLOW is an hybrid one composed of a structured region of highly stretched quadrilateral elements in the vicinity of solid boundaries and an unstructured region of triangular elements elsewhere (Fig. 4).

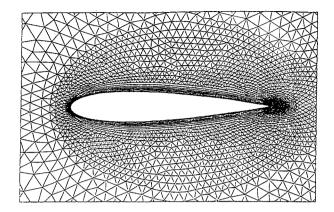


FIGURE 4 Computational mesh around NACA 0015 airfoil in Darrieus motion.

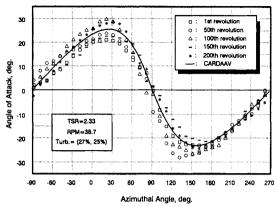


FIGURE 6 Angle of attack vs azimuthal angle at TSR=2.33 in turbulent wind.

RESULTS AND DISCUSSION

The prediction of the performance coefficient vs tip speed ratio (TSR) for Sandia 17-m wind turbine using the DMS model is given by Fig. 5. Comparison with vortex model (VDART3) (Strickland et al., 1980) and experimental data (Paraschivoiu, 1988) shows that the prediction by CARDAA code is well improved using CARDAAV and CARDAAX codes. Figs. 6 and 7 show the effect of atmospheric turbulence on the angle of attack and on the aerodynamic torque distribution. Unlike the periodic distribution predicted by CARDAAV, when turbulence is included the angle of attack distribution varies from one revolution to another. Furthermore, the ensemble-averaged aerodynamic torque distributions (Fig. 7) do not coincide with the periodic distribution. CARDAAS (1-D and 3-D) results predict well the experimental data given by Akins et al., (1987).

The simulation of the dynamic stall hysteresis loop using 3DVF code with indicial model is given by Fig. 8.

Compared to CARDAAV results, predictions in the upwind and downwind regions of the turbine are well compared to experimental data (Akins, 1989). The performance predictions at different tip speed ratio (Fig. 9) is also well predicted using 3DVF code.

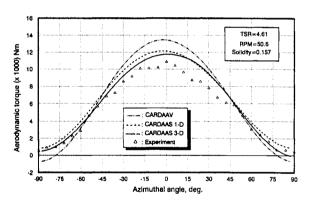


FIGURE 7 Aerodynamic torque at TSR=4.61 and turb.=(27%, 25%).

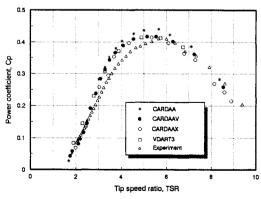


FIGURE 5 Performance coefficient vs tip speed ratio at RPM=50.6.

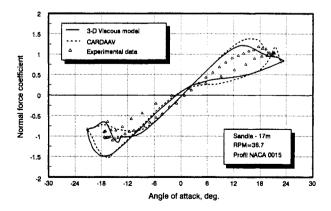


FIGURE 8 Normal force coefficient vs angle of attack at TSR=2.49.

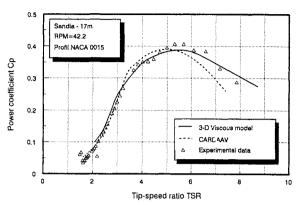


FIGURE 9 Power coefficient vs tip-speed ratio for Sandia 17-m.

In the above aerodynamic codes the model used for dynamic stall is based on semi-empirical methods. The simulation of the flow around an airfoil in Darrieus motion using Navier-Stokes solver is given by Fig. 10 in term of streamline evolution. Results using TKFLOW code can predict the region where the dynamic stall may occur.

CONCLUSION

Aerodynamic loads and performance of Darrieus wind turbine depend on the flowfield of the wind through the surface swept by the blades. The computer codes developed in this study represent a good tool for calculating the aerodynamic loads and performance of the vertical-axis wind turbines. Although the DMS models are still

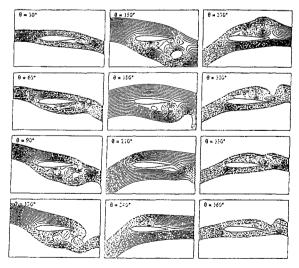


FIGURE 10 Computed streamlines around NACA 0015 airfoil in Darrieus motion.

being used in many places and are judged to be satisfactory because of its inexpensive approaches it is very important to include atmospheric turbulence into account especially for large wind turbines. In the case of wind turbine interferences such as wind farms the 3DVF seems to be more suitable since it can compute the flow velocity everywhere in the rotational plane as well as in its vicinity. The dynamic stall simulation using Navier-Stokes equations presents a good tool to predict the unsteady flow for Darrieus motion and more effort are underway to introduce a turbulence model in the present code.

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Fourrier coefficients

Nomenclature

 A_i^+, B_i^+

| c | : | blade chord, m |
|-------------------------|---|--|
| C_D | : | blade airfoil section drag coefficient |
| C_L | : | blade airfoil section lift coefficient |
| C_N | : | blade airfoil section normal-force coefficient |
| C_T | : | blade airfoil section tangential-force coefficient |
| N | ; | number of blades |
| r | ; | local rotor radius, m |
| Re_b | : | local Reynolds number |
| $S_{r,z,\theta,\omega}$ | : | source terms |
| TSR | : | tip speed ratio |
| <i>u</i> , <i>v</i> | : | components of the absolute velocity, m/s |
| u_f, v_f | : | fluctuations velocities, m/s |
| V_{abs} | : | absolute velocity, m/s |
| $V_{\mathbf{x}_i}$ | : | local ambient wind, velocity m/s |
| W | : | relative inflow velocity, m/s |
| X | : | tip speed ratio |
| α | : | angle of attack, deg. |
| θ | : | azimuthal angle, deg. |
| δ | : | meridional angle, deg. |
| ν | : | cinematic viscosity, m^2/s |
| ν_e | : | effective viscosity, m^2/s |
| ν_{t} | : | eddy viscosity, m^2/s |
| Ω | : | turbine rotational speed, s^{-1} |
| ρ | : | density, kg/m ³ |
| $	au^+$ | : | dimensionless time |
| η^+ | : | reduced frequency |
| | | |
| | | |

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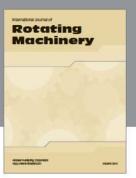
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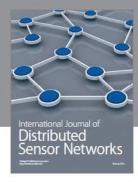
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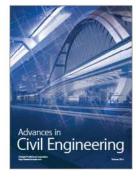








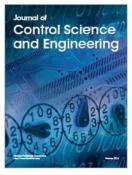




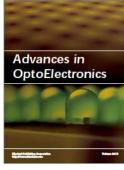




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